

Applications of Controllers and Compensators

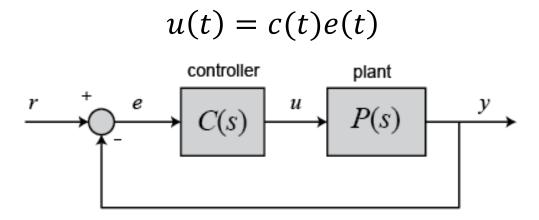
XMUT315 Control Systems Engineering

Topics

- Applications of controllers or compensators.
- Examples of applications of Proportional, Derivative,
 Integral controllers and their combinations.
- Examples of applications of Lag, Lead, and Lag-lead compensators.
- Practical circuit implementations of controllers or compensators.
- Tuning in of the controllers.

Applications of Controllers

- We will consider the following unity-feedback system.
- The output of the controller (u), which is equal to the control input to the plant, is calculated in the time domain from the feedback error (e) as follows:



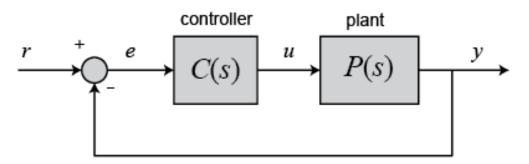
 First, let's take a look at how the controller works in a closed-loop system using the block diagram shown above.

Applications of Controllers

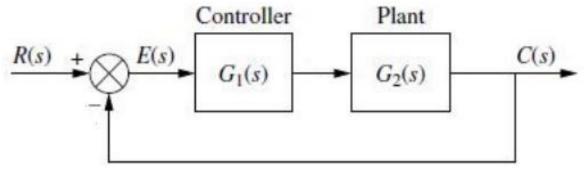
- The variable (e) represents the tracking error, the difference between the desired output (r) and the actual output (y).
- This error signal (e) is fed to the controller, and the controller computes this error signal parameters of controller with respect to time.
- Depending on the type of controller, these parameters are:
 - K_p for proportional controller,
 - K_i/s for integral controller,
 - $K_d s$ for derivative controller, or
 - any of their combinations such as: $K_p + K_i/s$ for PI controller, $K_p + K_d s$ for PD controller, and $K_p + K_i/s + K_d s$ for PID controller.

Applications of Controllers

- The control signal (u) to the plant is equal to the error times the magnitude of the parameters of the controller.
- This control signal (u) is fed to the plant and the new output
 (y) is obtained.
- The new output (y) is then fed back and compared to the reference to find the new error signal (e).
- The controller takes this new error signal and computes an update of the control input.
- This process continues while the controller is in effect.



• The goal of examples of controller applications is to show how each of the terms: K_p , K_i , and K_d , contributes to obtaining the common goals of fast rise time, minimal overshoot, and zero steady-state error.

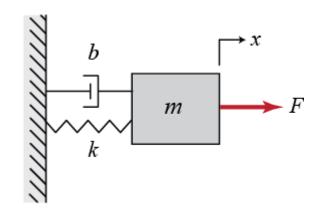


- The goals listed above are commonly found in the design specification of the control systems.
- We might have other (more specific) design specification depending on the needs and requirements.

Suppose we have a simple mass-spring-damper system. The governing equation of this system is:

$$m\left(\frac{d^2x}{dx^2}\right) + b\left(\frac{dx}{dt}\right) + kx = F$$

- a. Derive the transfer function of the system.
- b. If m = 1 kg, b = 10 N s/m, k = 20 N/m, and F = 1 N, determine the transfer function of the system.
- c. Simulate the step response of the open-loop system in MATLAB.
- d. Analyse the result of simulation in part (c) in terms of DC gain and steady-state error, rise time and settling time. What are characteristics of the controller needed to fix the problems?



 Taking the Laplace transform of the governing equation, we get:

$$ms^2X(s) + bsX(s) + kX(s) = F(s)$$

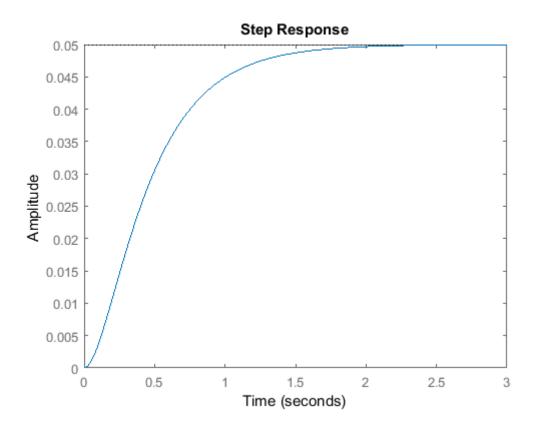
The transfer function between the input force and the output displacement then becomes:

$$\frac{X(s)}{F(s)} = \frac{1}{ms^2 + bs + k}$$

Let: m = 1 kg, b = 10 N s/m, k = 20 N/m, and F = 1 N. Substituting these values into the above transfer function:

$$\frac{X(s)}{F(s)} = \frac{1}{s^2 + 10s + 20}$$

b. Let's first view the open-loop step response.



- c. The DC gain of the plant transfer function is 1/20, so 0.05 is the final value of the output to a unit step input.
 - This corresponds to a steady-state error of 0.95, which is quite large.
 - Furthermore, the rise time is about 1 second and the settling time is about 1.5 seconds.
 - Let's design a controller that will reduce the rise time, reduce the settling time, and eliminate the steady-state error.

P Controllers

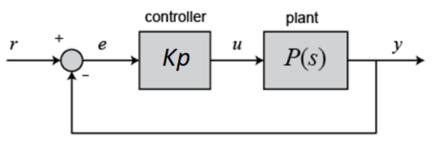
- For a proportional controller, the control signal (u) to the plant is equal to the proportional gain (K_p) times the magnitude of the error.
- The output of a proportional controller, which is equal to the control input to the plant, is calculated in the time domain from the feedback error as follows:

$$u(t) = K_p e(t)$$

 Thus, the transfer function of a proportional controller is found by taking the Laplace transform of system equation:

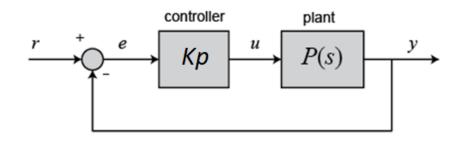
$$G_c(s) = K_p$$

Where: K_p = proportional gain.



P Controllers

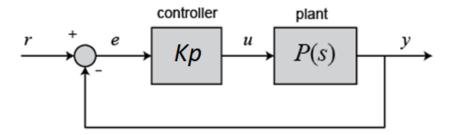
- Increasing the proportional gain (K_p) has the effect of proportionally increasing the control signal for the same level of error.
- The fact that the controller will "push" harder for a given level of error tends to cause the closed-loop system to react more quickly, but also to overshoot more.
- Another effect of increasing K_p is that it tends to reduce, but not eliminate, the steady-state error.



Proportional Controller

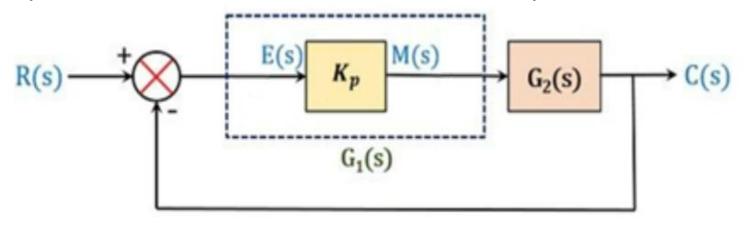
- Let's talk about the characteristics of the proportional controller.
- The proportional controller (K_p) reduces the rise time, increases the overshoot, and reduces the steady-state error.
- Transfer function equation of the proportional controller is:

$$G_c = K_p$$



Example for Proportional Controller

For the given simple mass-spring-damper system, add a proportional controller in series with the system.



- a. Derive the transfer function equation of the system.
- b. Using trial and error method, determine the appropriate value of the parameter of the controller. Then, simulate the transient response of the system in MATLAB.
- c. Analyse the response of the system in terms of rise time, overshoot, settling time and steady-state error.

Example for Proportional Controller

a. The transfer function equation of a proportional controller is:

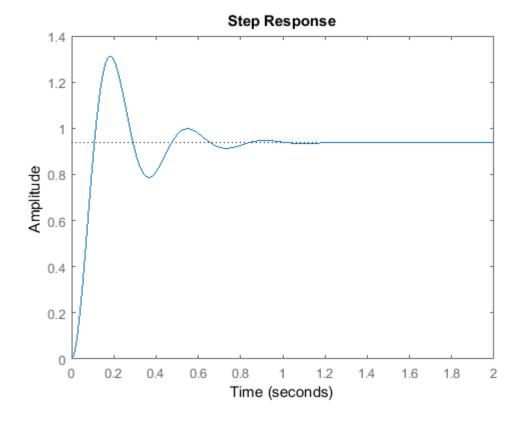
$$G_c(s) = K_p$$

The closed-loop transfer function of unity-feedback system with a proportional controller is as follow where X(s) is our output (equals Y(s)) and our reference R(s) is the input:

$$T(s) = \frac{X(s)}{R(s)} = \frac{G_c G_p}{1 + G_c G_p} = \frac{K_p \left(\frac{1}{s^2 + 10s + 20}\right)}{1 + K_p \left(\frac{1}{s^2 + 10s + 20}\right)}$$
$$= \frac{K_p}{s^2 + 10s + (20 + K_p)}$$

Example for Proportional Controller

- b. Let the proportional gain (K_p) equal 300. The following figure shows the step response of the example system with proportional controller.
- c. The plot below shows that the proportional controller reduced both the rise time and the steady-state error, increased the overshoot, and decreased the settling time by a small amount.



D Controller

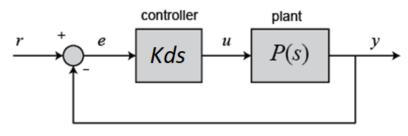
- For a derivative controller, the control signal (u) to the plant is equal to the derivative gain (K_d) times the derivative of the error.
- The output of a derivative controller, which is equal to the control input to the plant, is calculated in the time domain from the feedback error as follows:

$$u(t) = K_d \frac{de}{dt}$$

 Thus, the transfer function of a derivative controller is found by taking the Laplace transform of system equation:

$$G_c(s) = K_d s$$

Where: K_d = derivative gain.



D Controller

- The addition of a derivative term to the controller (K_d) adds the ability of the controller to "anticipate" error.
- With simple proportional control, if K_p is fixed, the only way that the control will increase is if the error increases.
- With derivative control, the control signal can become large
 if the error begins sloping upward, even while the magnitude
 of the error is still relatively small.

P(s)

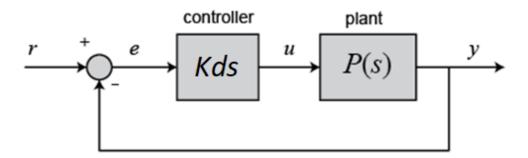
Kds

- This anticipation tends to add damping to the system,
 thereby decreasing overshoot.
- The addition of a derivative term, however, has no effect on the steady-state error.

PD Controller

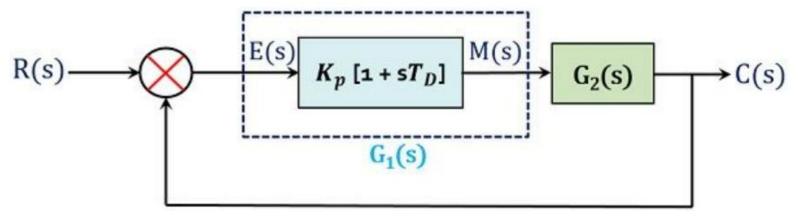
- Now, let's take a look at the characteristics of the PD controller.
- Addition of derivative control (K_d) tends to reduce both the overshoot and the settling time.
- Transfer function equation of the PD controller is:

$$G_c = K_p(1 + sT_d)$$



Example for PD Controller

For the given simple mass-spring-damper system, add a proportional-derivative controller in series with the system.



- a. Derive the transfer function equation of the system.
- b. Using trial and error method, determine the appropriate values of the parameters of the controller. Then, simulate the transient response of the system in MATLAB.
- c. Analyse the response of the system in terms of rise time, overshoot, settling time and steady-state error.

Example for PD Controller

a. The transfer function of the PD controller is (note: $K_d = K_p T_d$):

$$G_c(s) = K_d s + K_p$$

The closed-loop transfer function of the given system with a PD controller is:

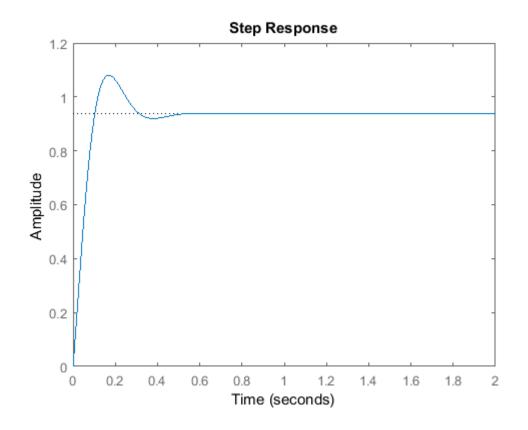
$$T(s) = \frac{X(s)}{R(s)} = \frac{G_c(s)G_p(s)}{1 + G_c(s)G_p(s)}$$

$$= \frac{\left(K_d s + K_p\right) \left(\frac{1}{s^2 + 10s + 20}\right)}{1 + \left(K_d s + K_p\right) \left(\frac{1}{s^2 + 10s + 20}\right)}$$

$$= \frac{K_d s + K_p}{s^2 + (10 + K_d)s + (20 + K_p)}$$

Example for PD Controller

- b. Let K_p equal 300 as before and let K_d equal 10.
- This plot shows that the addition of the derivative term reduced both the overshoot and the settling time and had a negligible effect on the rise time and the steady-state error.



I Controllers

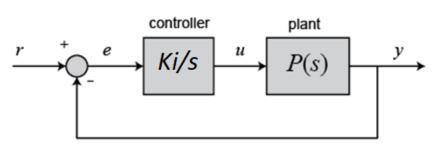
- For an integral controller, the control signal (u) to the plant is equal to the integral gain (K_i) times the integral of the error.
- The output of an integral controller, which is equal to the control input to the plant, is calculated in the time domain from the feedback error as follows:

$$u(t) = K_i \int e(t)dt$$

 Thus, the transfer function of an integral controller is found by taking the Laplace transform of system equation:

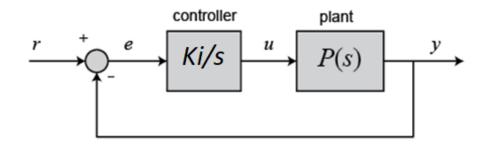
$$G_c(s) = \frac{K_i}{s}$$

Where: K_i = integral gain.



I Controllers

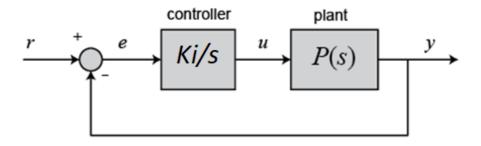
- The addition of an integral term to the controller (K_i) tends to help reduce steady-state error.
- If there is a persistent, steady-state error, the integrator builds and builds, thereby increasing the control signal and driving the error down.
- A drawback of the integral term, however, is that it can make the system more sluggish (and oscillatory) since when the error signal changes sign, it may take a while for the integrator to "unwind."



PI Controller

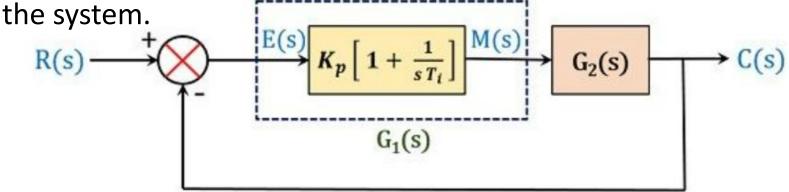
- Let's investigate the characteristics of the PI controller.
- Addition of integral control (K_i) tends to decrease the rise time, increase both the overshoot and the settling time, and reduces the steady-state error.
- Transfer function equation of PI controller is:

$$G_c(s) = K_p \left(1 + \frac{1}{sT_i} \right)$$



Example for PI Controller

For the given simple mass-spring-damper system in Tutorial for Example 1, add a proportional-integral controller in series with



- a. Derive the transfer function equation of the system.
- b. Using trial and error method, determine the appropriate values of the parameters of the controller. Then, simulate the transient response of the system in MATLAB.
- c. Analyse the response of the system in terms of rise time, overshoot, settling time and steady-state error.

Example for PI Controller

a. The transfer function of the PI controller is (note: $K_i = K_p/T_i$):

$$G_c(s) = K_i/s + K_p$$

For the given system, the closed-loop transfer function with a PI controller is:

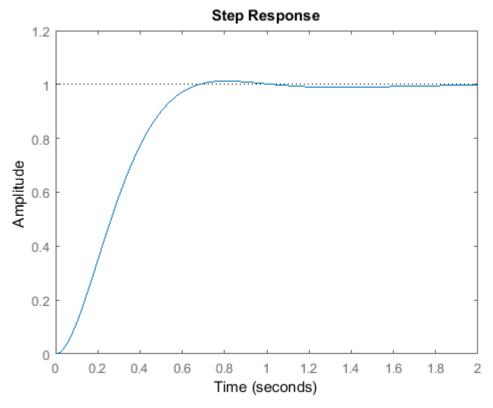
$$T(s) = \frac{X(s)}{R(s)} = \frac{G_c(s)G_p(s)}{1 + G_c(s)G_p(s)}$$

$$= \frac{\left(K_i/s + K_p\right)\left(\frac{1}{s^2 + 10s + 20}\right)}{1 + \left(K_i/s + K_p\right)\left(\frac{1}{s^2 + 10s + 20}\right)}$$

$$= \frac{K_p s + K_i}{s^3 + 10s^2 + (20 + K_p)s + K_i}$$

Example for PI Controller

- b. Let's reduce K_p to 30 and let K_i equal 70.
- c. We have reduced the proportional gain (K_p) because the integral controller also reduces the rise time and increases the overshoot as the proportional controller does (double effect).



The above response shows that the integral controller eliminated the steady-state error in this case.

PID Controllers

 For a PID controller, the output of a PID controller, which is equal to the control input to the plant, is calculated in the time domain from the feedback error as follows:

$$u(t) = K_p e(t) + K_i \int e(t)dt + K_d \frac{de}{dt}$$

 The transfer function of a PID controller is found by taking the Laplace transform of system equation:

$$G_c(s) = K_p + \frac{K_i}{s} + K_d s = \frac{K_d s^2 + K_p s + K_i}{s}$$

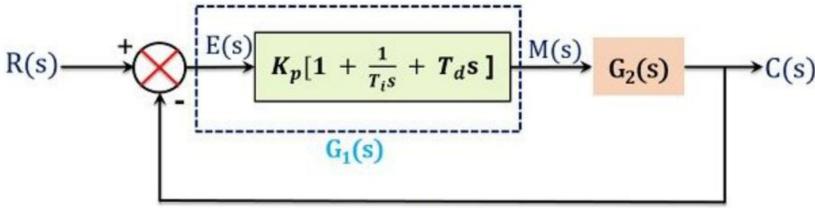
Where: K_p = proportional gain, K_i = integral gain, and K_d = derivative gain.

PID Controller

- Now, let's examine the characteristics of the PID control.
- PID controller tends to combine the characteristics of PI and PD controller. So, it is capable for improving both of the transient response and steady-state characteristics of the system.

Example for PID Controller

For the given simple mass-spring-damper system, add a proportional-integral-derivative controller in series with the system.



- a. Derive the transfer function equation of the system.
- b. Using trial and error method, determine the appropriate values of the parameters of the controller. Then, simulate the transient response of the system in MATLAB.
- c. Analyse the response of the system in terms of rise time, overshoot, settling time and steady-state error.

Example for PID Controller

a. Now, let's examine PID control. The transfer function of the PID controller is (note: $K_d = K_p T_d$ and $K_i = K_p / T_i$):

$$G_c(s) = K_d s + K_i/s + K_p$$

The closed-loop transfer function of the given system with a PID controller is:

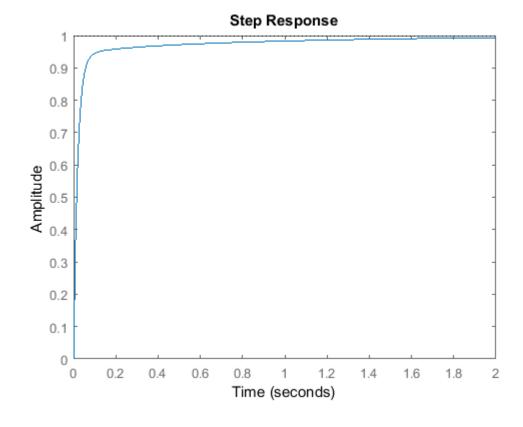
$$T(s) = \frac{X(s)}{R(s)} = \frac{G_c(s)G_p(s)}{1 + G_c(s)G_p(s)}$$

$$= \frac{\left(K_d s + K_i/s + K_p\right)\left(\frac{1}{s^2 + 10s + 20}\right)}{1 + \left(K_d s + K_i/s + K_p\right)\left(\frac{1}{s^2 + 10s + 20}\right)}$$

$$= \frac{K_d s^2 + K_p s + K_i}{s^3 + (10 + K_d)s^2 + (20 + K_p)s + K_i}$$

Example for PID Controller

- b. After several iterations of tuning, the gains $K_p = 350$, $K_i = 300$, and $K_d = 50$ provided the desired response.
- c. Now, we have designed a closed-loop system with no overshoot, fast rise time, and no steady-state error.



Summary of Controllers

• Summary of controllers:

Controller	Transfer Function	Characteristics	
Name	Equation		
Р	K_{p}	Reduces the rise time, increases the	
		overshoot, and reduces the steady-state error.	
I	K_i	Reduces steady-state error.	
	S		
D	K_ds	Increases the transient response	
		responsiveness and characteristics.	
PI	$K_p + K_i/s$	Decrease the rise time, increase both the	
		overshoot and the settling time, and reduces	
		the steady-state error.	
PD	$K_p + K_d s$	Reduce both the overshoot and the settling	
		time.	
PID	$K_d s^2 + K_p s + K_i$	Improve both of the transient response and	
	$\frac{\alpha}{S}$	steady-state characteristics.	

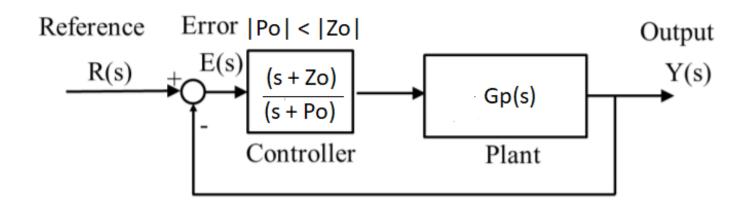
Summary of Controllers

- The general effects of each controller parameter : K_p , K_i , and K_d on a closed-loop system are summarized in the table below.
- Note, these guidelines hold in many cases, but not all.
- If you truly want to know the effect of tuning the individual gains, you will have to do more analysis, or will have to perform testing on the actual system.

Controller	Rise Time	Overshoot	Settling Time	Steady-State Error
K_p	Decrease	Increase	Small Change	Decrease
K_i	Decrease	Increase	Increase	Decrease
K_d	Small Change	Decrease	Decrease	No Change

Lag Compensator

 Lag compensator is commonly employed in the control systems to improve steady-state conditions but has little influence on the transient response of the systems.

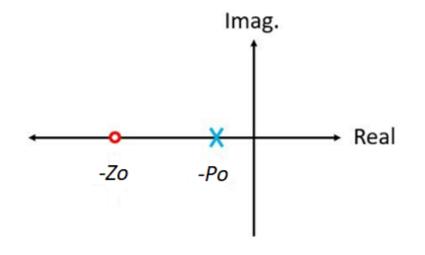


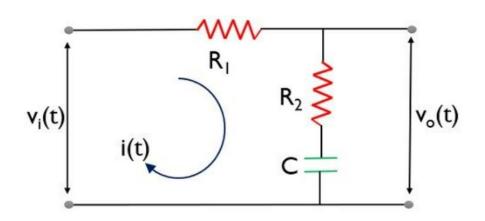
 Lag compensators reduce steady-state error, so sometimes we want smaller steady-state error rather than shorter rise and settling time as in a lead compensator.

Phase-lag compensator:

- The integrator in PI controller can cause some practical problems; e.g., "integrator windup" due to actuator saturation.
- PI controller is often approximated by "lag control."

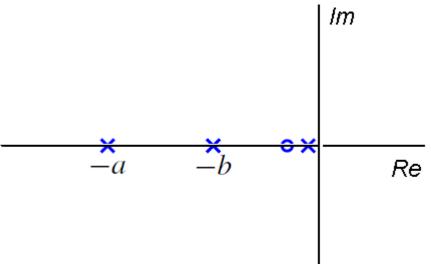
$$G_c(s) = \frac{(s - z_0)}{(s - p_0)}$$
 with $|p_0| < |z_0|$





- That is, the pole is closer to the origin than the zero.
- Because $|z_0|>|p_0|$, the phase "added to the open-loop transfer function is negative. . . "phase lag"
- Pole often placed very close to origin e.g., $p_0 \approx 0.01$ to minimize impact towards transient response performance.
- Zero is placed near pole. e.g., $z_0 \approx 0.1$.
- We want $|G_c(s)| \approx 1$ for all s to preserve transient response (and hence, have nearly the same placement of poles as for a proportional controller).
- Idea is to improve steady-state error but to modify the transient response as little as possible.

- That is, using proportional control, we have pole locations we like already, but poor steady-state error.
- So, we add a lag compensator to minimally disturb the existing good pole locations but improve steady-state error.
- Good steady-state error without overflow problems.
- Very similar to proportional control.



The uncompensated system had loop gain:

$$K(before) = \lim_{s \to 0} G(s)$$

The lag-compensated system has loop gain:

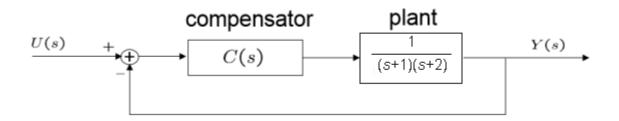
$$K(\text{after}) = \lim_{s \to 0} G_c(s)G(s) = \left(\frac{Z_0}{P_0}\right) \lim_{s \to 0} G(s)$$

- Since $|z_0|>|p_0|$, there is an improvement in the position or velocity or acceleration error constant of the system, and a reduction in steady-state error.
- Transient response is mostly unchanged, but slightly slower settling due to small-magnitude slow "tail" caused by lag compensator.

 The control system given below suffers from issues in both steady state and transient response conditions.

$$P(s) = \frac{1}{(s+1)(s+2)}$$
 and $C(s) = 1$

- Steady-state: non-zero steady-state error.
- Transient response: sluggish system that takes time to settle down.

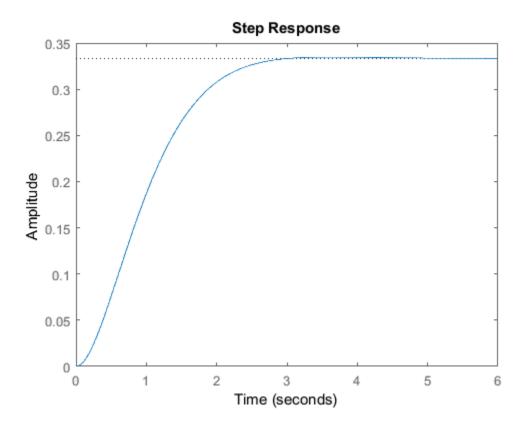


Example System for Lag Compensator

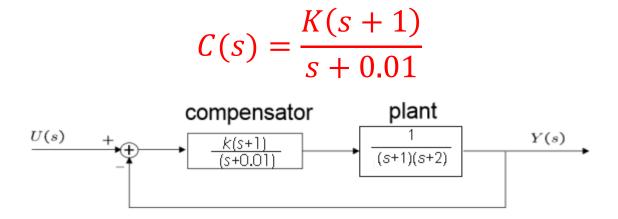
- a. Simulate the uncompensated system in MATLAB. Comment on the result of the simulation.
- b. Design a lead compensator that will be able to fix the problem observed in part (a).
- Simulate in MATLAB and compare the uncompensated and compensated systems. Observe whether the compensator has achieved its purpose.

Example System for Lag Compensator

a. Looking into the unit step response of the given system there are issues as highlighted before e.g. non-zero steady-state error and slow (sluggish) response of the system.



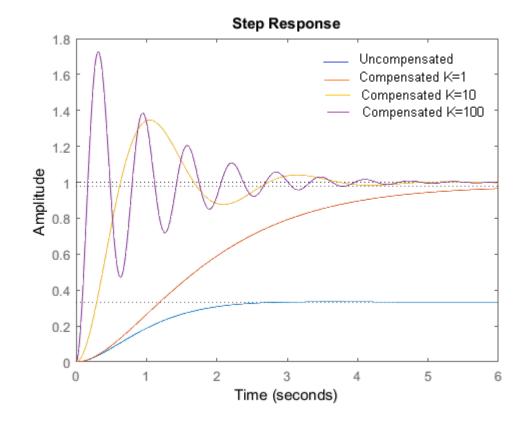
b. By trial and error, the gain and the pole and zero of the lead compensator (e.g. with K=1, 10, and 100) are determined:



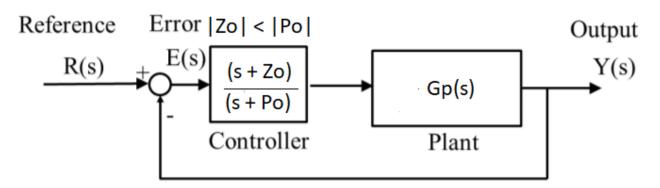
 Looking into the response of the compensated system, the plot shows smaller steady-state error than uncompensated system

Plots shown are with K = 1 (orange line), K = 10 (yellow line), and K = 100 (purple line).

Notice the growing oscillation as you increase the system gain (K), but settling time increases for all cases.



 Lead compensator is typically used in the control systems to improve the transient response and hence the stability of the systems.



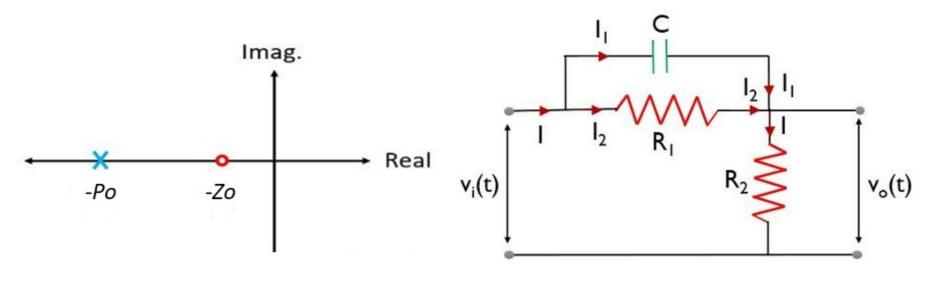
 The lead compensators improve transient response and stability, but they do not typically reduce steady-state error.

Phase-lead compensator:

- Derivative magnifies noise.
- Instead of D-control or PD-control use "lead control."

$$G_c(s) = \frac{(s - z_0)}{(s - p_0)}$$
 with $|z_0| < |p_0|$

That is, the zero is closer to the origin than the pole.

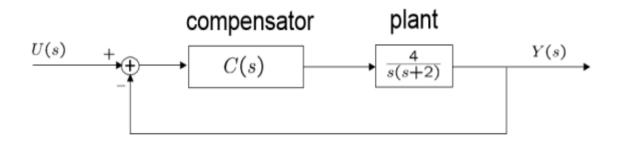


- Lead compensator is the same form as lag compensator, but with different intention:
 - Lag compensator does not change much since $P_0 \approx Z_0 \approx 0$. Instead, lag compensator improves steady-state error.
 - Lead compensator does change locus. Pole and zero locations chosen so that poles will pass through desired point $s=s_1$.

 The control system given below suffers from issues in the transient response conditions.

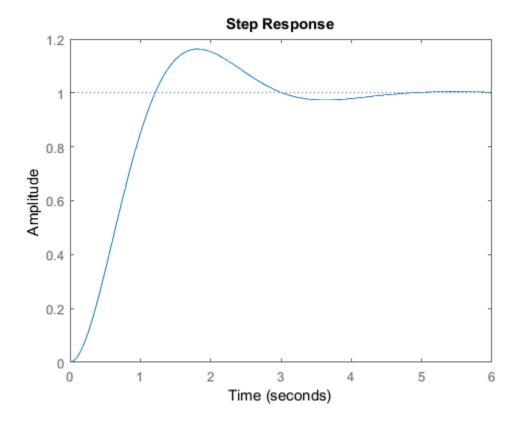
$$P(s) = \frac{4}{s(s+2)} \qquad \text{and} \qquad C(s) = 1$$

- Rise time: take some time for the system to rise up.
- Settling time: sluggish system that takes time to settle down.



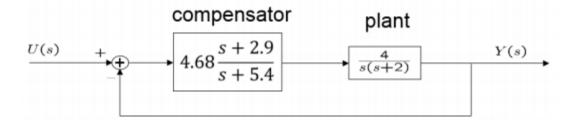
- a. Simulate the uncompensated system in MATLAB. Comment on the result of the simulation.
- b. Design a lead compensator that will be able to fix the problem observed in part (a).
- c. Simulate in MATLAB and compare the uncompensated and compensated systems. Observe whether the compensator has achieved its purpose.

a. Looking into the unit step response of the given system there are issues as highlighted before e.g. slow (sluggish) response of the system e.g. long rise time and settling time.



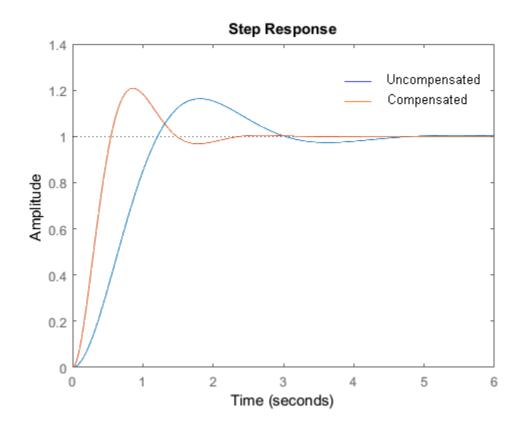
b. By trial and error, the gain and the pole and zero of the lead compensator are determined:

$$C(s) = \frac{4.68(s+2.9)}{s+5.4}$$

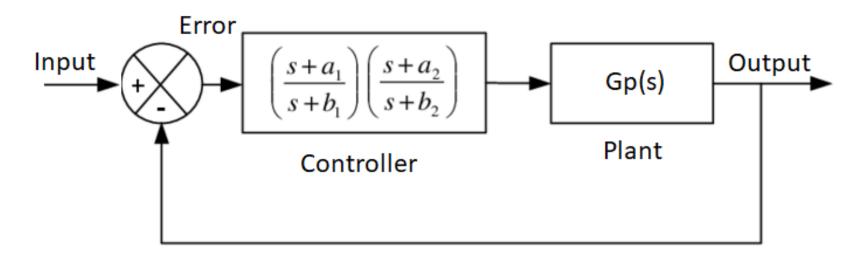


c. From the plot compensated system (red line) reaches steady state faster (shorter rise and settling times) than uncompensated system (blue line), although it has a higher percentage overshoot, M_p .

the step response plot of the uncompensated and compensated systems with lead compensator as shown in the figure below.



 For lead-lag compensator, it combines lead compensator and lag compensator.



 Lead-lag compensator provides the benefits of both lead and lag compensators e.g. improve performance in terms of steady-state conditions and transient responses.

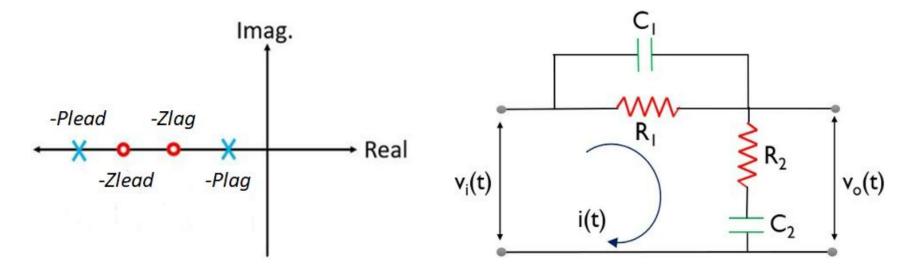
Lead-lag compensator:

The transfer function of the lead-lag compensator is:

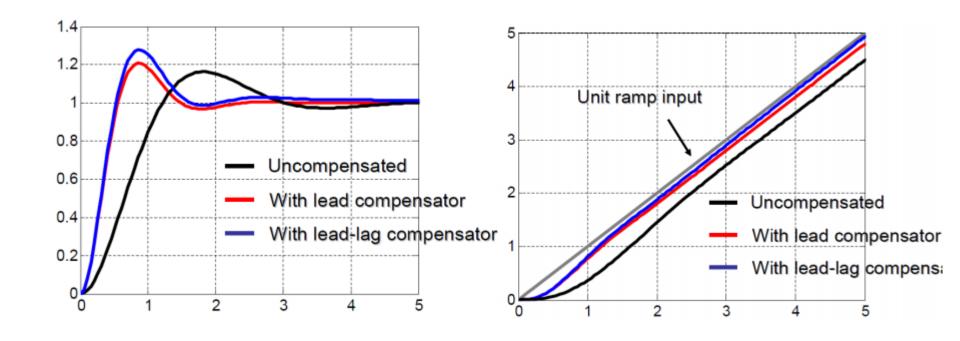
$$\frac{\left(s-z_{lag}\right)\left(s-z_{lead}\right)}{\left(s-p_{lag}\right)\left(s-p_{lead}\right)}$$

with

$$|p_{lag}| < |z_{lag}|$$
 and $|z_{lead}| < |p_{lead}|$



- For lead-lag compensator, left figure (step response), right figure (ramp response)
- Faster transient response when given step input (left).
- Smaller steady-state error when given ramp input (right)



- Lead-lag compensator improves transient response and steady-state condition.
- Design of the lead-lag compensator requires careful design of its individual parts e.g. lag compensator and lead compensator.
- Trial and error is typically employed to get the best set up for the lead-lag compensator.

If we must satisfy both the transient and steady-state specifications:

- 1. Design a lead compensator to meet transient specification first.
- 2. Include lead compensator with plant after its design is final.
- 3. Design a lag compensator (where "plant" = actual plant and lead compensator combined) to meet steady-state specification.

Summary of Compensator

• Summary of compensators:

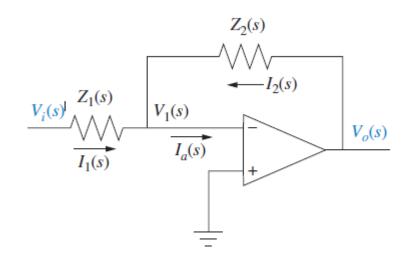
Name of	Transfer Function Equation	Characteristics
Compensator		
Lag	$\frac{(s-z_0)}{(s-p_0)}$ with $ p_0 < z_0 $	It improves steady-state error.
Lead	$\frac{(s-z_0)}{(s-p_0)}$ with $ z_0 < p_0 $	It improves transient response performance.
Lead-lag	$\frac{\left(s-z_{lag}\right)\left(s-z_{lead}\right)}{\left(s-p_{lag}\right)\left(s-p_{lead}\right)}$ with	It improves both steady-state error and transient response performance.
	$\left p_{lag} \right < \left z_{lag} \right $ and $\left z_{lead} \right < \left p_{lead} \right $	

Practical Implementations

- Practical implementation of controllers or compensators with op amp-based amplifier circuits.
- Practical implementations of controllers P, I, D and any of their combinations.
- Practical implementations of Lead, Lag, and Lead-lag compensators.

 We derived as the transfer function of an inverting operational amplifier whose configuration is shown above:

$$\frac{V_o(s)}{V_i(s)} = \frac{Z_2(s)}{Z_1(s)}$$



• By judicious choice of $Z_1(s)$ and $Z_2(s)$, this circuit is used as a building block to implement the compensators and controllers, such as PID controllers and lag-lead compensators using operational amplifiers.

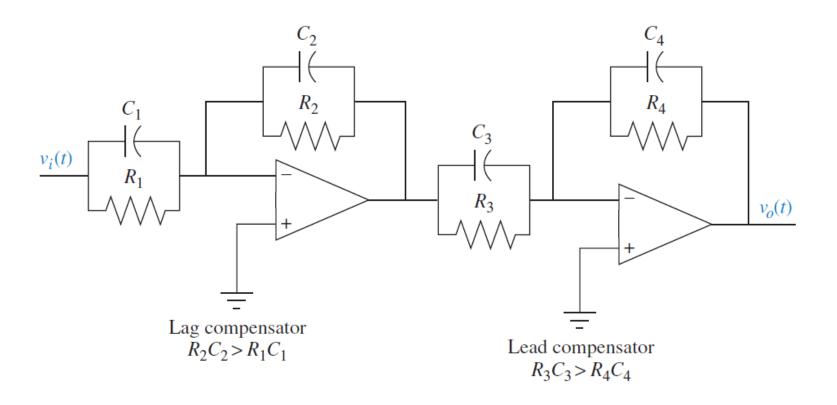
• Controller's active circuit realisations:

Function	$Z_1(s)$	$Z_2(s)$	$G_c(s) = \frac{Z_1(s)}{Z_2(s)}$
Proportional (gain)	R_1	R_2	$-\frac{R_1}{R_2}$
Integral	R	С	$-\frac{\left(\frac{1}{RC}\right)}{s}$
Derivative	С	R	-RCs
PI controller	R_1	R_2 and C (in series)	$-\left(\frac{R_1}{R_2}\right)\left(\frac{s+\frac{1}{R_2C}}{s}\right)$
PD controller	C and R_1 (in parallel)	R_2	$-R_2C\left(s+\frac{1}{R_1C}\right)$
PID controller	C_1 and R_1 (in parallel)	R_2 and C_2 (in series)	$-\left(\frac{R_2}{R_1} + \frac{C_1}{C_2} + R_2C_1s + \frac{\frac{1}{R_1C_2}}{s}\right)$

Compensator's active circuit realisations:

Function	$Z_1(s)$	$Z_2(s)$	$G_c(s) = \frac{Z_1(s)}{Z_2(s)}$
Lag compensator	C_1 and R_1 (in parallel)	C_2 and R_2 (in parallel)	$-\left(\frac{C_1}{C_2}\right)\left(\frac{s + \frac{1}{R_1C_1}}{s + \frac{1}{R_2C_2}}\right)$ where $R_2C_2 > R_1C_1$
Lead compensator	C_1 and R_1 (in parallel)	C_2 and R_2 (in parallel)	$-\left(\frac{C_1}{C_2}\right)\left(\frac{s + \frac{1}{R_1C_1}}{s + \frac{1}{R_2C_2}}\right)$ where $R_2C_2 < R_1C_1$
Lead-Lag compensator	Cascading lag compensator with lead compensator (as shown below).		

Cascading lag-lead compensator's active circuit realisations:



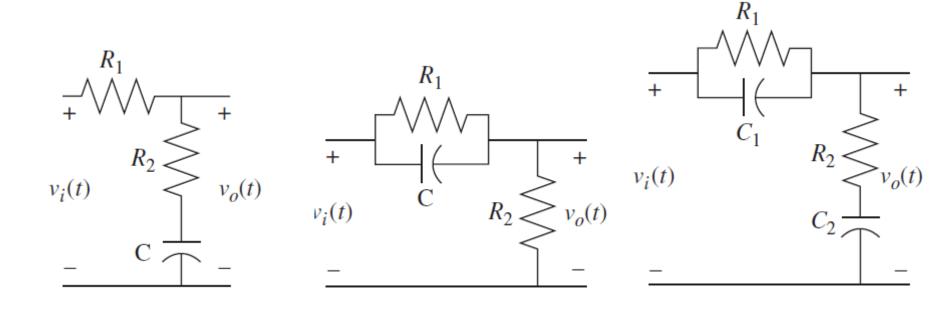
Practical Passive Implementations

Compensator's passive circuit realisations:

Function	$V_{\mathbf{i}}(\mathbf{s})$	$V_{o}(s)$	$G_c(s) = \frac{V_o(s)}{V_i(s)}$
Lag compensator	R_1, R_2 , and C (all in series)	C and R ₂ (both in series)	$\left(\frac{R_1}{R_1 + R_2}\right) \left(\frac{s + \frac{1}{R_2C}}{s + \frac{1}{(R_1 + R_2)C}}\right)$
Lead compensator	${\cal C}$ and ${\cal R}_1$ (both in parallel) and in series with ${\cal R}_2$	R_2	$\frac{s + \frac{1}{R_1 C}}{s + \frac{1}{R_1 C} + \frac{1}{R_2 C}}$
Lead-Lag compensator	C_1 and R_1 (both in parallel) and in series with C_2 and R_2 (both in series)	C ₁ and R ₂ (both in series)	$\frac{\left(s + \frac{1}{R_1 C_1}\right) \left(s + \frac{1}{R_2 C_2}\right)}{s + \left(\frac{1}{R_1 C_1} + \frac{1}{R_2 C_2} + \frac{1}{R_2 C_1}\right) s + \frac{1}{R_1 R_2 C_1 C_2}}$

Practical Passive Implementations

Compensator's passive circuit realisations:



Lag compensator

Lead compensator

Lag-lead compensator

Example of Practical Implementation

 Implement the PID controller when the transfer function of the PID controller is:

$$G_c(s) = \frac{(s+55.92)(s+0.5)}{s}$$

The transfer function of the controller can be put in the form:

$$G_c(s) = s + 56.42 + \frac{27.96}{s} = \left(\frac{R_2}{R_1} + \frac{C_1}{C_2}\right) + R_2C_1s + \frac{\frac{1}{R_1C_2}}{s}$$

Equating the transfer function of the controller with the PID controller:

$$s + 56.42 + \frac{27.96}{s} = \left(\frac{R_2}{R_1} + \frac{C_1}{C_2}\right) + R_2C_1s + \frac{\frac{1}{R_1C_2}}{s}$$

Example of Practical Implementation

 Comparing the PID controller in the table with the controller, we obtain the following three relationships:

$$\frac{R_2}{R_1} + \frac{C_1}{C_2} = 56.42$$
 $R_2 C_1 = 1$ $1/R_2 C_1 = 27.96$

• Since there are four unknowns and three equations, we arbitrarily select a practical value for one of the elements.

• Selecting $C_2 = 0.1 \,\mu\text{F}$, the remaining

off.

values are found to be $R_1=357.65~\mathrm{k}\Omega$, $R_2=178,891~\mathrm{k}\Omega$, and $R_1=5.59~\mathrm{\mu}$.

• The complete circuit is shown in the figure below, where the circuit element values have been rounded

- In majority of the cases, compensator is commonly designed for tackling a particular issue or problem in control system with its specific set up or arrangement.
- On the other hand, controller is typically intended and designed to be able to be adjusted or tuned-in to manage the operation of the system.
- There are many approaches to configure controllers. But these are typically classified into e.g. ad-hoc (on the spot), experimentation, or prescriptive formulas.

 There are various ways to configure and to tune in the controller in control system.

Method	Advantages	Disadvantages
Manual tuning	No math required.	Requires experience.
Software tools	Consistent tuning, can employ computer- automated control system design techniques, may include devices analysis, allows simulation before implementation, and can support non-steady-state (NSS) tuning.	Some cost or training involved.
Ziegler–Nichols	Proven method.	Process upset, some trial-and- error, very aggressive tuning.
Tyreus-Luyben	Proven method.	Process upset, some trial-and- error, very aggressive tuning.
Cohen–Coon	Good process models.	Some math required and only good for first-order processes.
Åström-Hägglund	Can be used for auto tuning; amplitude is minimum, so this method has lowest process upset.	The process itself is inherently oscillatory.

General Tips for Designing a PID Controller

When you are designing a PID controller for a given system, follow the steps shown below to obtain a desired response.

- Obtain an open-loop response and determine what needs to be improved.
- 2. Add a proportional control to improve the rise time.
- 3. Add a derivative control to reduce the overshoot.
- 4. Add an integral control to reduce the steady-state error.
- 5. Adjust each of the gains K_p , K_i , and K_d until you obtain a desired overall response. You can always refer to the table shown to find out which controller controls which characteristics.

- Lastly, please keep in mind that you do not need to implement all three controllers (proportional, derivative, and integral) into a single system, if not necessary.
- For example, if a PI controller meets the given requirements (like the above example), then you don't need to implement a derivative controller on the system.
- Keep the controller as simple as possible.